

Homework 5 Solutions

#3-11

a). Assume that the  $x$  axis in figure P3-11 is the principal axis for  $\epsilon_1$  and the  $y$  axis is the principal axis for  $\epsilon_2$ , with  $\epsilon_2 > \epsilon_1$ . Then from the first of Eqs (2-30), for a strain gage with no mounting error (i.e.  $\beta = 0^\circ$ ):

$$\epsilon_{xx'} = \frac{\epsilon_1 + \epsilon_2}{2} + \frac{\epsilon_1 - \epsilon_2}{2} \cos(2\theta)$$

whereas with a mounting error of  $\beta$ ,

$$\tilde{\epsilon}_{xx'} = \frac{\epsilon_1 + \epsilon_2}{2} + \frac{\epsilon_1 - \epsilon_2}{2} \cos[2(\theta + \beta)]$$

$\therefore$  the error,  $E$ , is

$$E = \tilde{\epsilon}_{xx'} - \epsilon_{xx'}$$

$$E = \frac{\epsilon_1 - \epsilon_2}{2} \{ \cos[2(\theta + \beta)] - \cos(2\theta) \}$$

But  $\cos \alpha - \cos \gamma = -2 \sin\left(\frac{\alpha + \gamma}{2}\right) \sin\left(\frac{\alpha - \gamma}{2}\right)$

$$\therefore \cos[2(\theta + \beta)] - \cos(2\theta) = -2 \sin(2\theta + \beta) \sin \beta$$

so that  $E$  can be written as

$$(1) \quad E = -(\epsilon_1 - \epsilon_2) \sin \beta \sin(2\theta + \beta)$$

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The percent error is

$$E' = \frac{\bar{E}_{xx} - E_{xx}}{E_{xx}} 100$$

$$E' = \frac{(E_1 - E_2) \{ \cos[2(\theta + \beta)] - \cos(2\theta) \}}{(E_1 + E_2) + (E_1 - E_2) \cos(2\theta)} 100$$

$$(2) \quad E' = \frac{\cos[2(\theta + \beta)] - \cos(2\theta)}{\frac{r+1}{r-1} + \cos(2\theta)} 100$$

$$r = E_1/E_2$$

Note that  $\cos[2(\theta + \beta)] = \cos(2\theta)\cos(2\beta) - \sin(2\theta)\sin(2\beta)$   
and  $\therefore$  for small values of  $\beta$ ,

$$\cos[2(\theta + \beta)] = \cos(2\theta) - 2\beta \sin(2\theta)$$

Thus  $E$  and  $E'$  become,

$$(3) \quad E = -(E_1 - E_2)\beta \sin(2\theta)$$

$$(4) \quad E' = \frac{-2\beta \sin(2\theta)}{\frac{r+1}{r-1} + \cos(2\theta)} 100$$

b).  $\beta, \theta, \epsilon_1/\epsilon_2$

c). Consider values of  $\theta$  in the approximate range of  $0^\circ$  to  $90^\circ$ .  
 The strain given in Eq (1) takes on a minimum magnitude when  $\sin(2\theta + \beta) = 0$  and a maximum magnitude when  $\sin(2\theta + \beta) = 1$

$\therefore E_{\min}$  occurs at  $\theta = -\frac{\beta}{2}$  or  $\theta = -\frac{\beta}{2} + \frac{\pi}{2}$

$E_{\max}$  occurs at  $\theta = -\frac{\beta}{2} + \frac{\pi}{4}$

Thus the strain gage orientations minimum and maximum error differ by  $45^\circ$ .

Note that as  $\beta \rightarrow 0$  (i.e. mounting error  $\rightarrow 0$ )

$E_{\min}$  occurs at  $\theta \rightarrow 0$  or  $\theta \rightarrow \frac{\pi}{2}$

$E_{\max}$  occurs at  $\theta \rightarrow \frac{\pi}{4}$

In other words, the strain gage reading is least sensitive to mounting errors when mounted along one of the principal axes and most sensitive when mounted at an angle of  $45^\circ$  relative to the principal axes.

This can also be readily seen from Eq (3). Note that if  $\epsilon_1 = \epsilon_2$  then  $E = 0$  for any  $\theta$  and any  $\beta$ .

Problem 3-11 part d  
 The error is given by Eq. (1) as

$$E = -(\epsilon_1 - \epsilon_2) \cdot \sin(\beta) \cdot \sin(2\theta + \beta)$$

For uniaxial strain state,

$$\epsilon_2 = -\nu \epsilon_1 = -\nu \cdot \epsilon$$

Therefore,

$$E = -\epsilon \cdot (1 + \nu) \cdot \sin(\beta) \cdot \sin(2\theta + \beta)$$

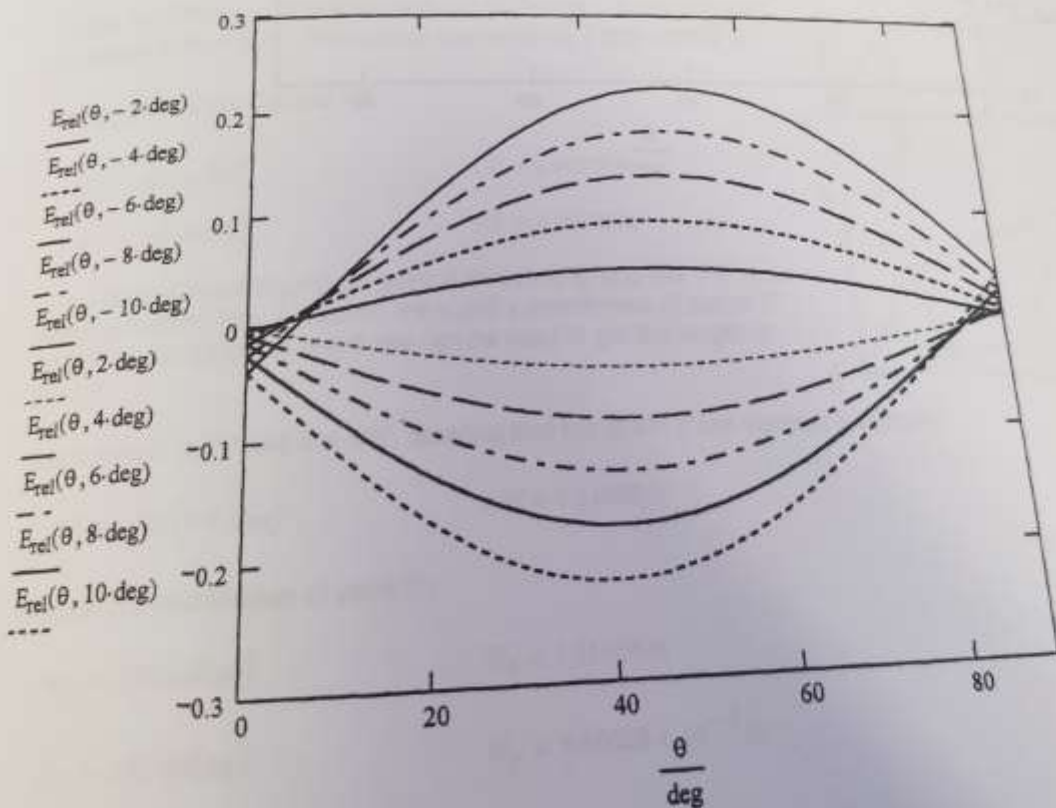
and

$$E_{rel} = \frac{E}{\epsilon} = \frac{(1 + \nu)}{2} \cdot \sin(\beta) \cdot \sin(2\theta + \beta)$$

With  $\nu = 0.285$

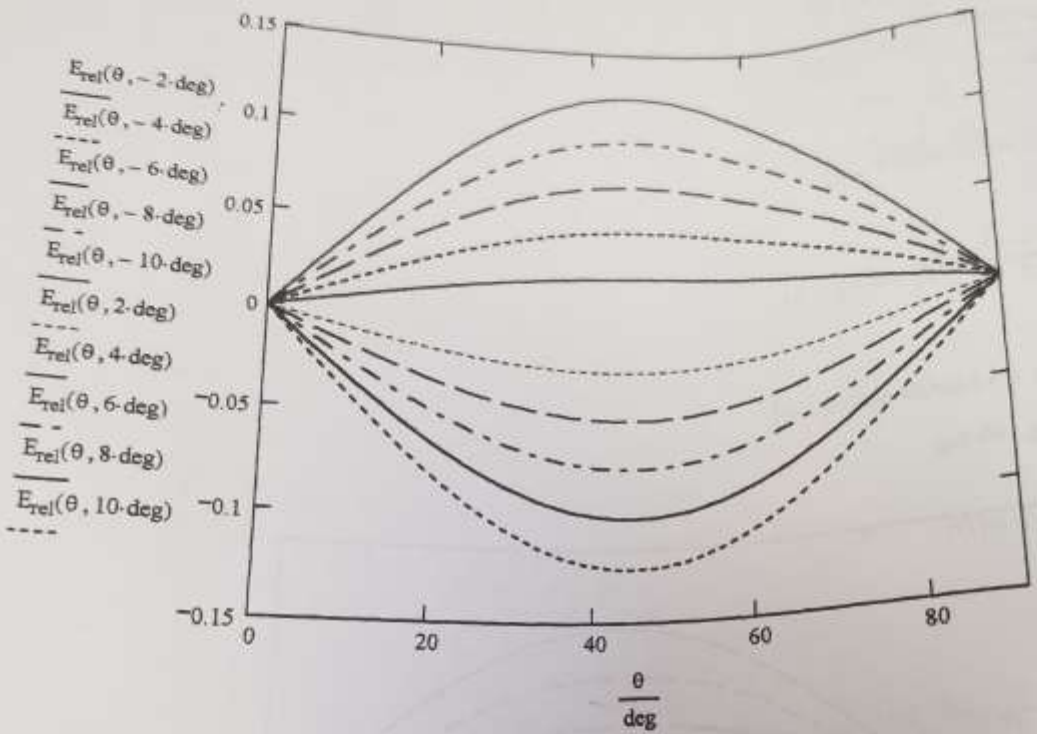
$$E_{rel}(\theta, \beta) = -(1 + \nu) \cdot \sin(\beta) \cdot \sin(2\theta + \beta)$$

$\theta := 0\text{-deg}, 1\text{-deg}, \dots, 90\text{-deg}$



If Eq. (3) is used instead,

$$E_{rel}(\theta, \beta) := -(1 - \nu) \cdot \beta \cdot \sin(2 \cdot \theta)$$



Problem 3-12  
input data:

$$AC := 1.2 \text{ in} \quad CD := 1.4 \text{ in} \quad AB := 1.4038 \text{ in} \quad B'D' := 1.203 \text{ in} \quad \theta := 22.7 \text{ deg}$$

$$\Delta\phi := .003 \quad \Delta\theta := .0023$$

Get strain  $\epsilon_{xx}$ :

$$AB := CD \quad AB = 1.4 \text{ in}$$

$$\epsilon_{xx} := \frac{AB' - AB}{AB} \quad \epsilon_{xx} = 2.71429 \times 10^{-3}$$

Get strain  $\epsilon_{x'x'}$ :

$$A'C' := B'D' \quad A'C' = 1.203 \text{ in}$$

$$\epsilon_{x'x'} := \frac{A'C' - AC}{AC} \quad \epsilon_{x'x'} = 2.5 \times 10^{-3}$$

To get the strain  $\epsilon_{yy}$ , drop a perpendicular from point C to the x axis, as shown in the figure. The vertical intersects the x axis at point E.

Get length of lines CE and AE:

$$CE := AC \cdot \sin(\theta) \quad CE = 0.46309 \text{ in}$$

$$AE := AC \cdot \cos(\theta) \quad AE = 1.10705 \text{ in}$$

In the deformed configuration, point E moves to E' and line C'E' is no longer a vertical line. However, the x and y coordinates of points C' and E' can be obtained and these can be used to get the length of line C'E'.

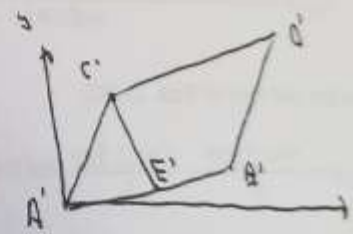
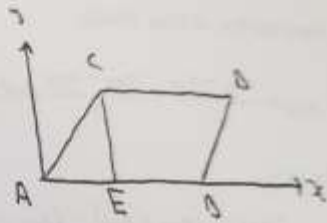
Get length of deformed line A'E', recalling that the strain in the element is uniform:

$$A'E' := AE \cdot (1 + \epsilon_{xx}) \quad A'E' = 1.11005 \text{ in}$$

Get x and y coordinates of point E':

$$E'_x := A'E' \cdot \cos(\Delta\phi) \quad E'_x = 1.11004 \text{ in}$$

$$E'_y := A'E' \cdot \sin(\Delta\phi) \quad E'_y = 5.55023 \times 10^{-3} \text{ in}$$





Get x and y coordinates of point C:

$$AC = BD$$

$$AC = 1.203 \text{ in}$$

$$C_x = AC \cdot \cos(\theta + \Delta\theta)$$

$$C_x = 1.10874 \text{ in}$$

$$C_y = AC \cdot \sin(\theta + \Delta\theta)$$

$$C_y = 0.4668 \text{ in}$$

Get length of line C'E:

$$CE' = \sqrt{(C_x - E_x)^2 + (C_y - E_y)^2} \quad CE' = 0.46125 \text{ in}$$

Get strain  $\epsilon_{yy}$ :

$$\epsilon_{yy} = \frac{CE' - CE}{CE}$$

$$\epsilon_{yy} = -3.97194 \times 10^{-3}$$

From the first of Eqs. (3-30):

$$\epsilon_{x'x'} = \frac{\epsilon_{xx} + \epsilon_{yy}}{2} + \frac{\epsilon_{xx} - \epsilon_{yy}}{2} \cdot \cos(2\theta) + \frac{1}{2} \cdot \gamma_{xy} \cdot \sin(2\theta)$$

Solve for  $\gamma_{xy}$ :

$$\gamma_{xy} = \frac{2 \cdot \epsilon_{x'x'} - (\epsilon_{xx} + \epsilon_{yy}) - (\epsilon_{xx} - \epsilon_{yy}) \cdot \cos(2\theta)}{\sin(2\theta)}$$

$$\gamma_{xy} = 2.19501 \times 10^{-3}$$

Use the last two of Eqs. (3-30):

$$\epsilon_{y'y'} = \frac{\epsilon_{xx} + \epsilon_{yy}}{2} - \frac{\epsilon_{xx} - \epsilon_{yy}}{2} \cdot \cos(2\theta) - \frac{1}{2} \cdot \gamma_{xy} \cdot \sin(2\theta)$$

$$\epsilon_{y'y'} = -3.75766 \times 10^{-3}$$

$$\gamma_{x'y'} = -(\epsilon_{xx} - \epsilon_{yy}) \cdot \sin(2\theta) + \gamma_{xy} \cdot \cos(2\theta)$$

$$\gamma_{x'y'} = -3.21954 \times 10^{-3}$$

Is Eq. (3-52) satisfied?

$$\epsilon_{xx} + \epsilon_{yy} - (\epsilon_{x'x'} + \epsilon_{y'y'}) = 0$$

Problem 3-13

From Problem 3-12:

$$\epsilon_{xx} = .00271$$

$$\epsilon_{yy} = -.00397$$

$$\gamma_{xy} = .00230$$

The principal strains are given by Eqs. (3-42):

$$\epsilon_1 = \frac{\epsilon_{xx} + \epsilon_{yy}}{2} + \sqrt{\left(\frac{\epsilon_{xx} - \epsilon_{yy}}{2}\right)^2 + \left(\frac{1}{2}\gamma_{xy}\right)^2}$$

$$\epsilon_1 = 2.886 \times 10^{-3}$$

$$\epsilon_2 = \frac{\epsilon_{xx} + \epsilon_{yy}}{2} - \sqrt{\left(\frac{\epsilon_{xx} - \epsilon_{yy}}{2}\right)^2 + \left(\frac{1}{2}\gamma_{xy}\right)^2}$$

$$\epsilon_2 = -4.146 \times 10^{-3}$$

The principal directions are given by Eq. (3-43):

$$\tan(\theta) = \frac{\epsilon - \epsilon_{xx}}{\frac{1}{2}\gamma_{xy}}$$

Therefore,

$$\theta_1 := \text{atan2}\left(\frac{1}{2}\gamma_{xy}, \epsilon_1 - \epsilon_{xx}\right)$$

$$\theta_1 = 9.114 \text{ deg}$$

$$\theta_2 := \text{atan2}\left(\frac{1}{2}\gamma_{xy}, \epsilon_2 - \epsilon_{xx}\right)$$

$$\theta_2 = -80.886 \text{ deg}$$



#3-14

$$u(x,y) = a_0 + a_1 x + a_2 y + a_3 xy$$

$$v(x,y) = a_4 + a_5 x + a_6 y + a_7 xy$$

Therefore, with  $E_{ij} = (i-1) - (j-1)$ ,

$$E_x = \frac{\partial u}{\partial x} = a_1 + a_3 y$$

$$E_y = \frac{\partial v}{\partial y} = a_6 + a_7 x$$

$$\gamma_{xy} = \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} = a_2 + a_3 x + a_5 + a_7 y$$

In matrix form these become,

$$\begin{pmatrix} E_x \\ E_y \\ \gamma_{xy} \end{pmatrix} = \begin{bmatrix} 0 & 1 & 0 & y & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & x \\ 0 & 0 & 1 & x & 0 & 1 & 0 & y \end{bmatrix} \begin{pmatrix} a_0 \\ a_1 \\ a_2 \\ a_3 \\ a_4 \\ a_5 \\ a_6 \\ a_7 \end{pmatrix}$$

Note that  $\tau_i = y_i = y_j = \tau_i = 0$  and  $\tau_j = x_k, y_k = y_j$

$$u_i = a_0$$

$$v_i = a_4$$

$$u_j = a_0 + a_1 x_j$$

$$v_j = a_4 + a_5 x_j$$

or, in matrix form,

$$\begin{pmatrix} u_i \\ v_i \\ u_j \\ v_j \\ u_k \\ v_k \\ u_l \\ v_l \end{pmatrix} = \begin{pmatrix} 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 \\ 1 & x_j & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & x_j & 0 \\ 1 & x_j & y_k & x_j y_k & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & x_j & y_k & x_j y_k \\ 1 & 0 & y_k & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} a_0 \\ a_1 \\ a_2 \\ a_3 \\ a_4 \\ a_5 \\ a_6 \\ a_7 \end{pmatrix}$$

These can be rearranged to give

$$\begin{pmatrix} u_i \\ u_j \\ u_k \\ u_l \\ v_i \\ v_j \\ v_k \\ v_l \end{pmatrix} = \begin{pmatrix} 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & x_j & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & x_j & y_k & x_j y_k & 0 & 0 & 0 & 0 \\ 1 & 0 & y_k & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & x_j & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & x_j & y_k \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \end{pmatrix} \begin{pmatrix} a_0 \\ a_1 \\ a_2 \\ a_3 \\ a_4 \\ a_5 \\ a_6 \\ a_7 \end{pmatrix}$$

These equations are of the form,

$$\begin{Bmatrix} \{u\}_{1, \text{node}} \\ \{u\}_{2, \text{node}} \end{Bmatrix} = \begin{bmatrix} [x]_{\text{node}} & [0] \\ [0] & [x]_{\text{node}} \end{bmatrix} \begin{Bmatrix} \{a\}_{1, \text{node}} \\ \{a\}_{2, \text{node}} \end{Bmatrix}$$

where  $\{u\}_{1, \text{node}} = [u_i \ u_j \ u_k \ u_l]^T$

$$\{u\}_{2, \text{node}} = [v_i \ v_j \ v_k \ v_l]^T$$

$$[x]_{\text{node}} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 1 & x_j & 0 & 0 \\ 1 & x_j & y_k & x_j y_k \\ 1 & 0 & y_k & 0 \end{bmatrix}$$

$$\{a\}_{1, \text{node}} = [a_0 \ a_1 \ a_2 \ a_3]^T$$

$$\{a\}_{2, \text{node}} = [a_4 \ a_5 \ a_6 \ a_7]^T$$

$$\{a\}_{1, \text{node}} = [x]_{\text{node}}^{-1} \{u\}_{1, \text{node}}$$

$$\{a\}_{2, \text{node}} = [x]_{\text{node}}^{-1} \{u\}_{2, \text{node}}$$

$$\text{or } \begin{Bmatrix} \{a\}_{node} \\ \{a\}_{node} \end{Bmatrix} = \begin{Bmatrix} [x]_{node}^{-1} & [0] \\ [0] & [x]_{node}^{-1} \end{Bmatrix} \begin{Bmatrix} \{u\}_{node} \\ \{u\}_{node} \end{Bmatrix}$$

$[x]_{node}^{-1}$  is computed as

$$[x]_{node}^{-1} = \frac{1}{x_j y_k} \begin{bmatrix} x_j y_k & 0 & 0 & 0 \\ -y_k & y_k & 0 & 0 \\ -x_j & 0 & 0 & x_j \\ 1 & -1 & 1 & -1 \end{bmatrix}$$

Note that  $x_j y_k$  is the area of the rectangular element. With this,

$$\begin{Bmatrix} a_0 \\ a_1 \\ a_2 \\ a_3 \\ a_4 \\ a_5 \\ a_6 \\ a_7 \end{Bmatrix} = \frac{1}{x_j y_k} \begin{bmatrix} x_j y_k & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ -y_k & y_k & 0 & 0 & 0 & 0 & 0 & 0 \\ -x_j & 0 & 0 & x_j & 0 & 0 & 0 & 0 \\ 1 & -1 & 1 & -1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & x_j y_k & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & -y_k & y_k & 0 & 0 \\ 0 & 0 & 0 & 0 & -x_j & 0 & 0 & x_j \\ 0 & 0 & 0 & 0 & 1 & -1 & 1 & -1 \end{bmatrix} \begin{Bmatrix} u_i \\ u_j \\ u_k \\ u_l \\ v_i \\ v_j \\ v_k \\ v_l \end{Bmatrix}$$

The nodal displacement vector is now rearranged to show to the original order,  $u_i, v_i, u_j, v_j$ , etc.

$$\begin{Bmatrix} a_0 \\ a_1 \\ a_2 \\ a_3 \\ a_4 \\ a_5 \\ a_6 \\ a_7 \end{Bmatrix} = \frac{1}{\tau_j y_{jk}} \begin{bmatrix} \tau_j y_{jk} & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ -y_{jk} & 0 & y_{jk} & 0 & 0 & 0 & 0 & 0 \\ -\tau_j & 0 & 0 & 0 & 0 & 0 & -\tau_j & 0 \\ 1 & 0 & -1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & \tau_j y_{jk} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & -y_{jk} & 0 & y_{jk} & 0 & 0 \\ 0 & 0 & 0 & -\tau_j & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} u_i \\ v_i \\ u_j \\ v_j \\ u_k \\ v_k \\ u_l \\ v_l \end{Bmatrix}$$

This is of the form,

$$\{a\} = [c] \{U_{node}\}$$

where  $\{a\} = [a_0 \ a_1 \ a_2 \ a_3 \ a_4 \ a_5 \ a_6 \ a_7]^T$

$$\{U_{node}\} = [u_i \ v_i \ u_j \ v_j \ u_k \ v_k \ u_l \ v_l]^T$$

$$[c] = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$[c]^{-1} = \frac{1}{x_j y_k} \begin{bmatrix} x_j y_k & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & x_j y_k & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & x_j y_k & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & x_j y_k & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & x_j y_k & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & x_j y_k & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & x_j y_k & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & x_j y_k \end{bmatrix}$$

$$\{e\} = [x][c]^{-1} \{u_{node}\}$$

where

$$[x] = \begin{bmatrix} 0 & 1 & 0 & y & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 1 & x \\ 0 & 0 & 1 & x & 0 & 1 & 0 & y \end{bmatrix}$$