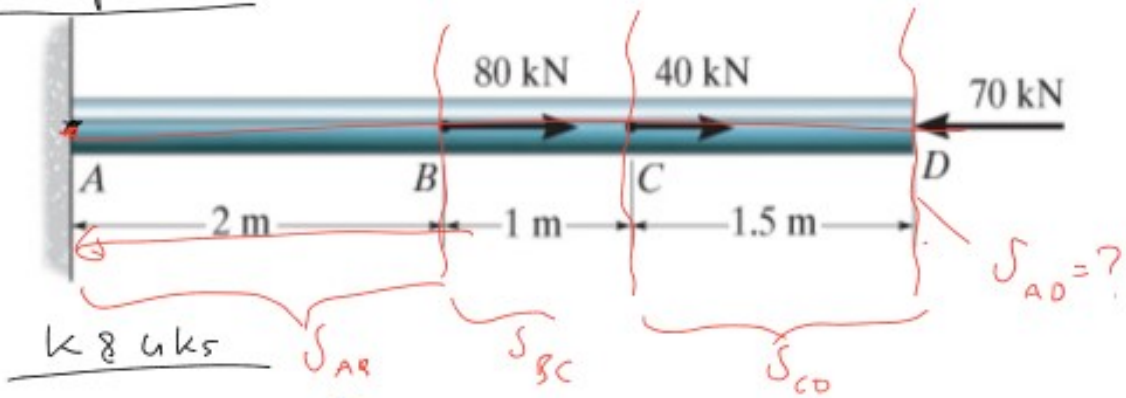


Lecture 1 Examples

Example 1



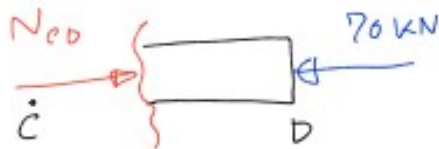
$E = 200 \text{ GPa}$
 $k: 70 \text{ kN} \pm, 40 \text{ kN} \pm, 80 \pm, d = 50 \dots$
 Dimensions
 A-36 Steel

Uks: $\delta_D = ?$, $\delta_{BC} = ?$, $N_{AB} = ?$

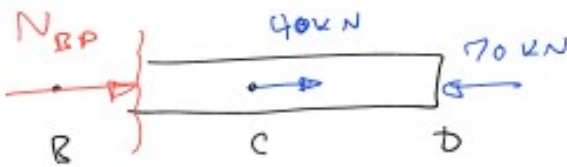
$$\delta = \sum \frac{P \cdot L}{A \cdot E}$$

FBD

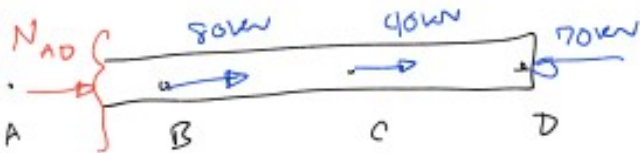
Equ. 1



$$\sum F_x: N_{CD} = 70 \text{ kN (C)}$$

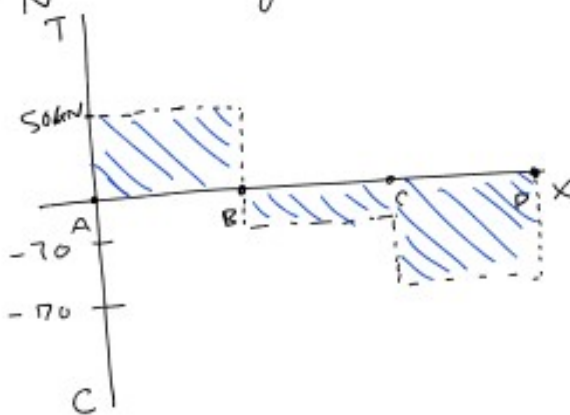


$$\sum F_x: N_{BC} = 30 \text{ kN (C)}$$



$$\sum F_x: N_{AB} = -50 \text{ kN (T)}$$

Normal Force Diagram



$$c) \delta_D = ?$$

$$\delta_D = \sum \frac{P \cdot L}{AE} = \frac{N_{CD} L_{CD}}{AE} + \frac{N_{BC} L_{BC}}{AE} + \frac{N_{AB} L_{AB}}{AE}$$

$$\text{Tables} \rightarrow A-36 \rightarrow E = 200 \text{ GPa}$$

$$A = \pi r^2 = \frac{\pi}{4} (d)^2 = \pi (25 \text{ mm})^2$$

$$\delta_D = \frac{[-70 (10^3) \text{ N}] [1.5 (10^3) \text{ mm}]}{\pi (25 \text{ mm})^2 [200 (10^3) \text{ N/mm}^2]}$$

$$+ \frac{[-30 (10^3) \text{ N}] [1 (10^3) \text{ mm}]}{\pi (25 \text{ mm})^2 [200 (10^3) \text{ N/mm}^2]}$$

$$+ \frac{[50 (10^3) \text{ N}] [2 (10^3) \text{ mm}]}{\pi (25 \text{ mm})^2 [200 (10^3) \text{ N/mm}^2]}$$

$$\delta_D = -89.1 (10^{-3}) \text{ mm}$$

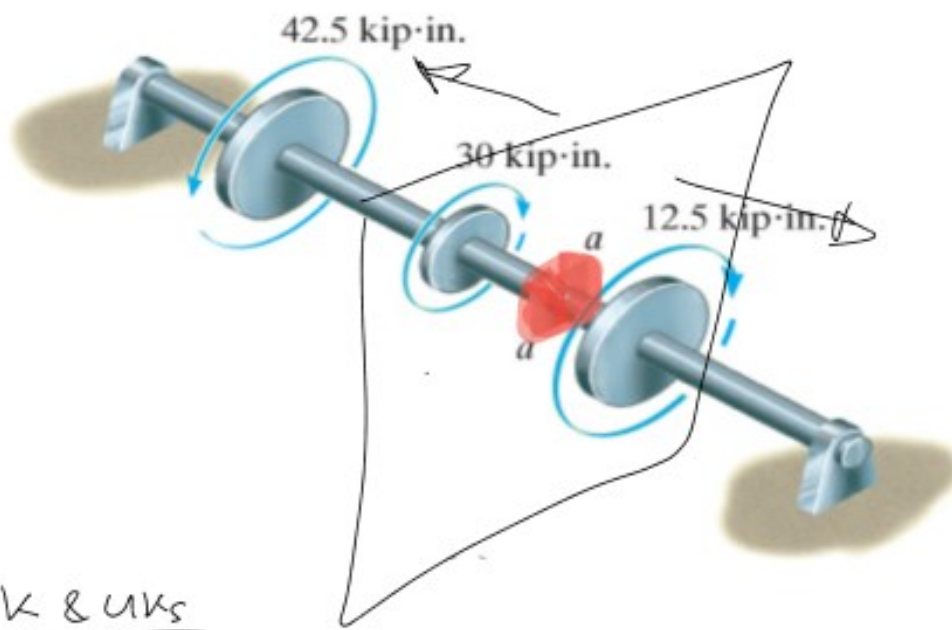
$$b) \delta_{BC} = ?$$

$$\delta_{BC} = -76.4 (10^{-3}) \text{ mm}$$

$$\sigma = \frac{F}{A}$$

$$F = \sigma \cdot A$$

$$\sum F = 0$$

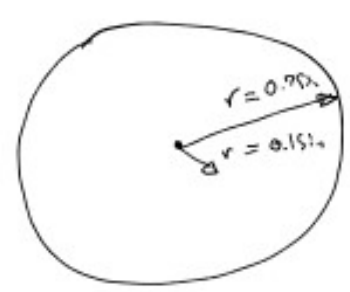
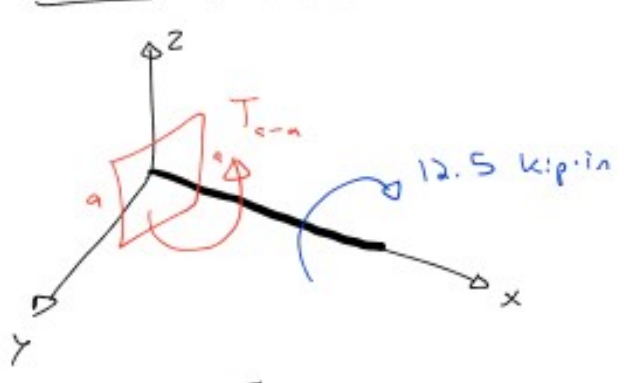


K & UKS

k: T_1, T_a, T_3 $d = 1.5 \text{ in}$,

UKS: $\tau_a = ?$ $\tau_b = ?$ $\rightarrow \tau = \frac{T \cdot \rho}{J}$ $\rightarrow T_{\text{on } a} = ? , J = ?$

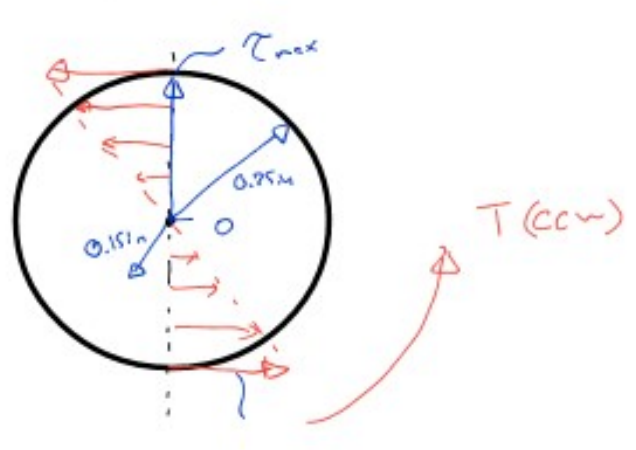
FRD Method Sections



$$\rightarrow \sum M_x : -12.5 + T = 0$$

$$T = 12.5 \text{ kip-in ccw}$$

Stress Distribution Diagram



Torsional Shear Stress

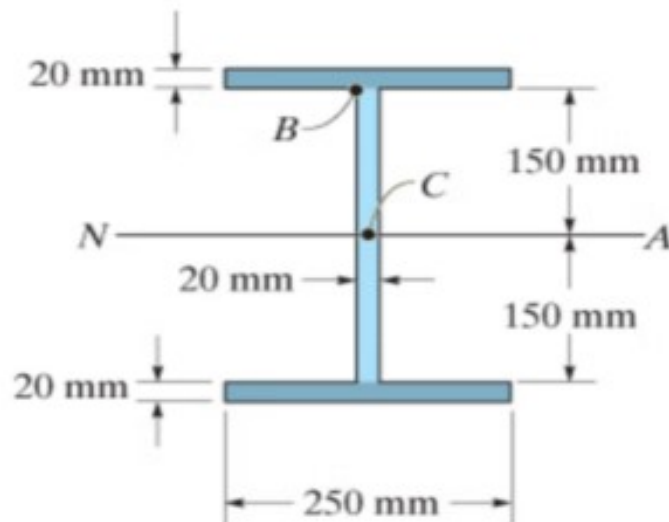
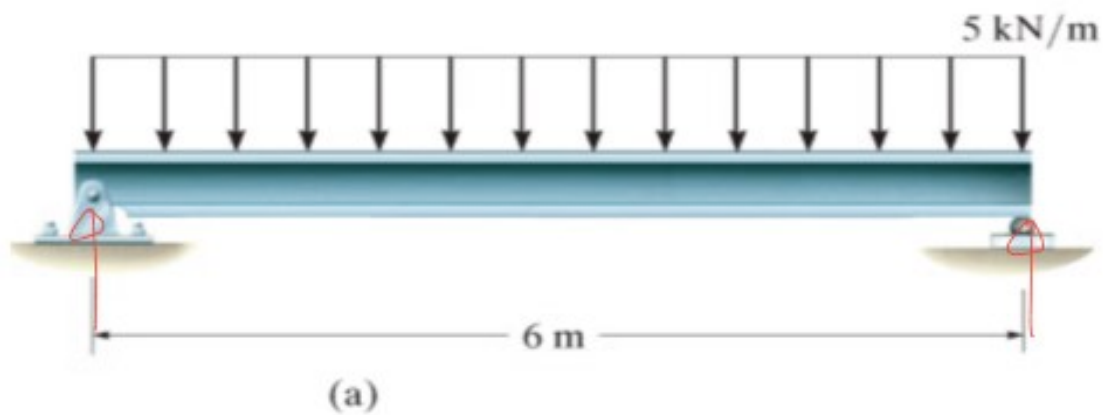
$$\tau = \frac{T \cdot \rho}{J} = \begin{cases} \tau_a = \frac{T_{e-a} \cdot \rho_a}{J} \\ \tau_b = \frac{T_{e-a} \cdot \rho_b}{J} \end{cases}$$

$$J = \frac{\pi}{2} r^4 = \frac{\pi}{2} (0.75 \text{ in})^4 = 0.497 \text{ in}^4$$

$$\tau_a = \frac{(12.5 \text{ kip}\cdot\text{in})(0.75 \text{ in})}{\text{"}} = 18.9 \text{ ksi}$$

$$\tau_b = \frac{(\text{"})(0.15 \text{ in})}{\text{"}} = 3.77 \text{ ksi}$$

Determine the absolute maximum bending stress in the beam and draw the stress distribution over the cross section at this location. Also what is the stress at point B?



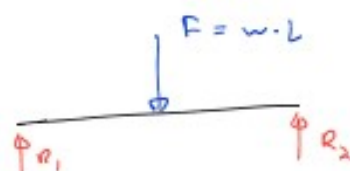
K & UKs

K: Load, Support, Dimensions

UKs: $R_1, R_2,$ $\sigma_{max} = ? = \frac{M(x) \cdot c}{I}$ $M_{max} = ?$

$M(x)$ Diagram \rightarrow $V(x)$ Diagram

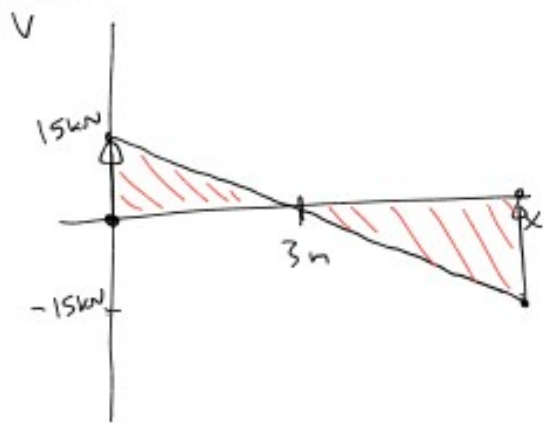
FBD



Equil

$$R_1 = R_2 = 15 \text{ kN}$$

Shear Force Diagram



$$\Delta V = F_0$$

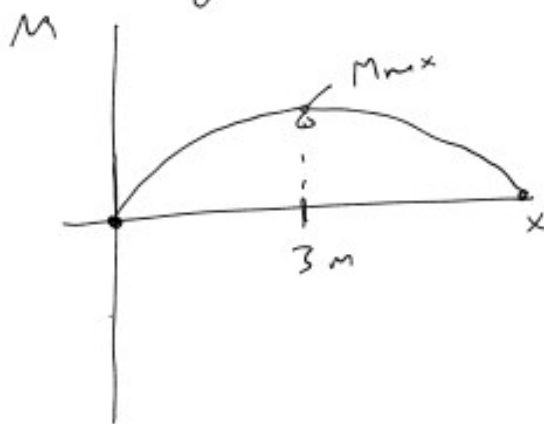
$$\frac{dV}{dx} = w(x)$$

$$V = \int w(x) dx$$

$$V(x) = \int -5 dx + c = -5x + c$$

$$V(x) = -5x + 15$$

Bending Moment Diagram



$$\frac{dM}{dx} = V(x)$$

$$M = \int V(x) dx + C$$

$$M = \int -5x + 15 dx + C$$

$$M(x) = -\frac{5x^2}{2} + 15x + C$$

$$M(0) = 0 + 0 + C$$

$$C = 0$$

$$M_{max} \Rightarrow \frac{dM}{dx} = V = 0$$

$$x = 3m \quad M(3) = M_{max} = -\frac{5(3)^2}{2} + 15(3) = 22.5 \text{ kN}\cdot\text{m}$$

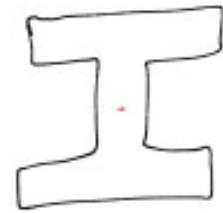
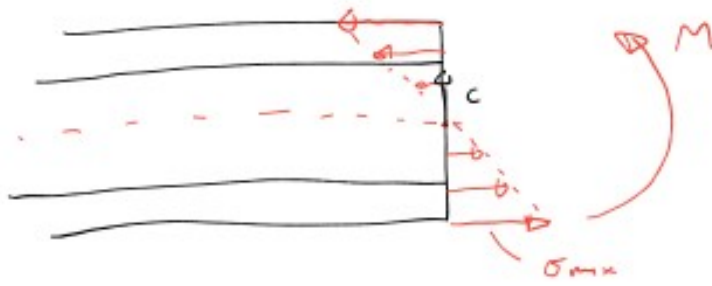
Abs Maximum Bending Stress, σ_{max}

$$\sigma_{max} = \frac{M_{max} \cdot c}{I}$$

We search the tables to find I

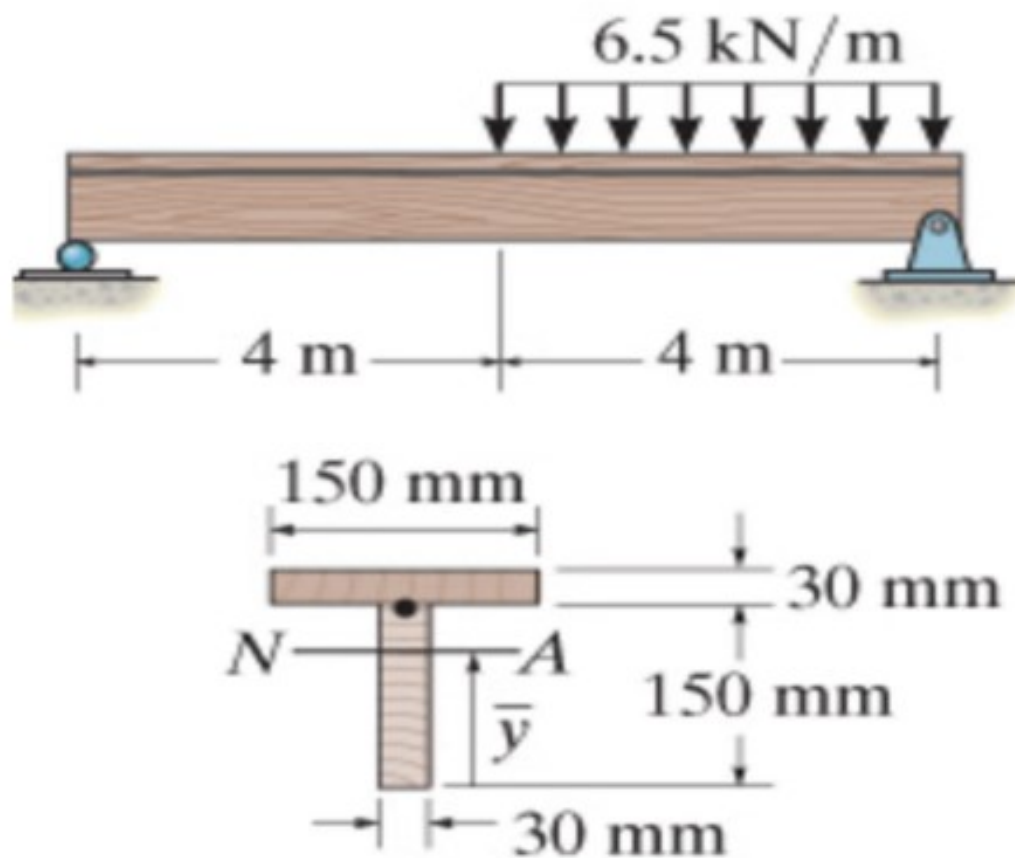
$$I = 301.3 \text{ E-6 m}^4$$

Stress Distribution



$$\sigma_{max} = \frac{M_{max} \cdot c}{I} = 12.7 \text{ MPa}$$

The Beam is made of two boards. Determine the maximum shear stress in the glue necessary to hold the boards together along the seam where they are joined.



K & Uks

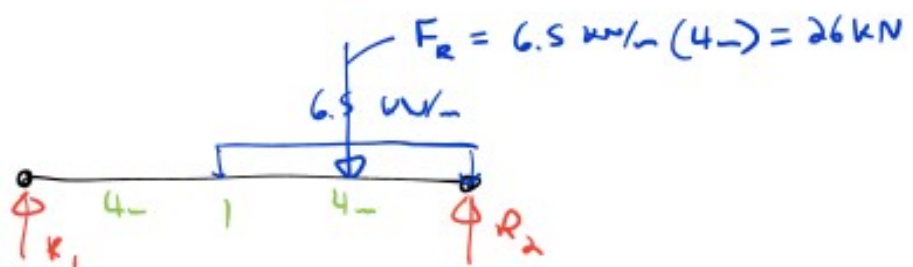
type of support, dimensions, $w = 6.5 \text{ kN/m}$

$\bar{y} = ?$, $\tau_{\text{max, glue}} = ?$, $V(x) = ?$, $Q = ?$, $I = ?$, $t = ?$

Reactions ✓?

$$\tau_{\text{max, glue}} = \frac{V_{\text{max}} Q}{I \cdot t}$$

FBD

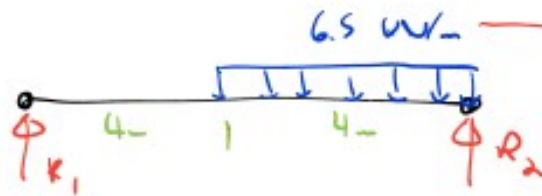


$\Sigma F \ \& \ \Sigma M$

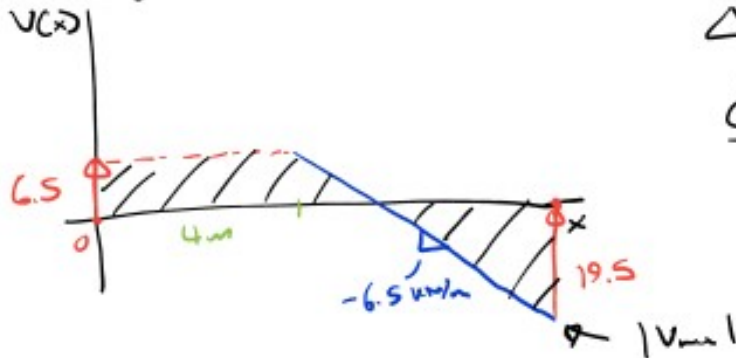


$$R_1 = 6.5 \text{ kN}$$

$$R_2 = 19.5 \text{ kN}$$



Show Force Diagram



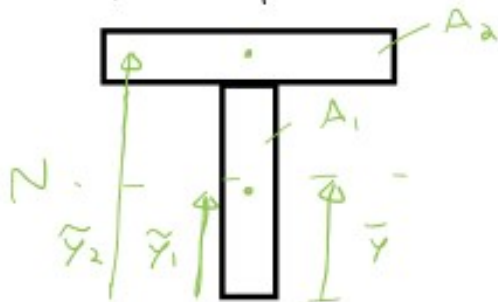
$$\Delta V = F_c$$

$$\frac{dV}{dx} = w(x)$$

Find, I the 2nd Moment of Inertia

a) Find \bar{y}

Centroid, Neutral Axis - \bar{y}



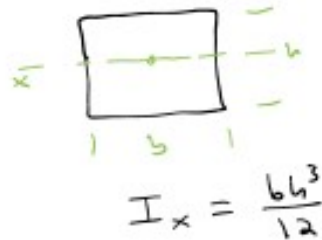
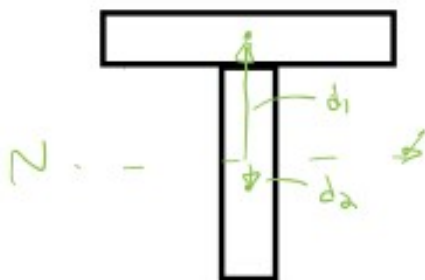
$$\bar{y} = \frac{\sum \bar{y} A}{\sum A}$$

$$\bar{y} = \frac{\bar{y}_1 A_1 + \bar{y}_2 A_2}{A_1 + A_2}$$

$$\bar{y} = 0.12 \text{ m}$$

b) Find, I

The I of a Single Rectangle



$$I_x = \frac{b^3}{12}$$

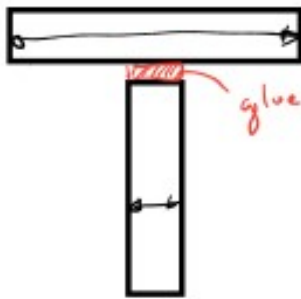
Parallel Axis theorem

$$I = \sum (I_x + A d^2)$$

$$I = (I_{x1} + A_1 d_1^2) + (I_{x2} + A_2 d_2^2)$$

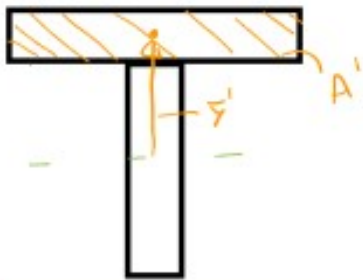
$$I = 27E-6 \text{ m}^4$$

Find, $t = ?$



$$t = 0.03 \text{ m}$$

Find, $Q = ?$



$$\tau = \frac{VQ}{I +}$$

the top of the beam is held in place by the bottom. So Q should be taken from the top because it will produce a larger, τ !

$$Q = \bar{y}' A'$$

center of A'
from the neutral axis

Surface Area
of Interest

$$Q = 0.2025 \text{ E-3 m}^3$$

Solve for $\tau_{max, glue}$

$$\tau_{max, glue} = \frac{V_{max} Q}{I +}$$

$$\tau_{max, glue} = 4.88 \text{ MPa}$$