

1.22	ALLOY STEEL FIG. 1.3 & 1.4	STRUCTURAL STEEL FIG. 1.5
YIELD POINT	N/A	265 MPa
YIELD STRENGTH	450 MPa	N/A
UPPER YIELD POINT	N/A	280 MPa
LOWER YIELD POINT	N/A	265 MPa
MODULUS OF RESILIENCE	N/A	0.1855 MPa
ULTIMATE TENSILE STRENGTH	715 MPa	470 MPa
STRAIN AT FRACTURE	0.23	0.26
PER CENT ELONGATION	23%	26%

1.24 ASSUME: 1. PLANE SECTIONS NORMAL TO THE AXIS OF THE BAR REMAIN PLANE UNDER APPLICATION OF THE LOAD.
2. HOOKE'S LAW APPLIES.

EQUILIBRIUM:
 $P = \sigma A \quad (a)$

HOOKE'S LAW:
 $\sigma = E \epsilon \quad (c)$

CONTINUITY:
 $\Delta L = \int_0^L \epsilon \, dx \quad (b)$

GEOMETRY
 $A(x) = b(d_0 - \frac{x}{L}(d_0 - d_1)) \quad (d)$

SUB (d) INTO (a)

$$\sigma = \frac{P}{b(d_0 - \frac{x}{L}(d_0 - d_1))} \quad (e)$$

SUB. (c) & (e) INTO (b):

$$\Delta L = \int_0^L \frac{P}{Eb} \left[d_0 - \frac{x}{L}(d_0 - d_1) \right]^{-1} dx$$

$$\Delta L = \frac{PL}{Eb} \int_0^L \frac{dx}{d_0 L - x(d_0 - d_1)} = \frac{PL}{Eb} \left(\frac{1}{d_0 - d_1} \right) \ln \frac{d_0}{d_1}$$

1.25

$$\text{RODS: } A_R = 4 \left(\frac{\pi}{4} 15^2 \right) = 706.9 \text{ mm}^2$$

$$\text{PIPE: } A_P = \frac{\pi}{4} (100^2 - 90^2) = 1492 \text{ mm}^2$$

$$\text{AFTER ASSEMBLY: } T_R = C_P = 4(65) = 260 \text{ kN}$$

$$\sigma_R = \frac{4(65000)}{706.9} = 367.8 \text{ MPa}$$

$$\sigma_P = \frac{-4(65000)}{1492} = -174.3 \text{ MPa}$$

AFTER PRESSURE IS APPLIED:

EQUILIBRIUM:

$$P \left(\frac{\pi}{4} 90^2 \right) = \Delta T_R + \Delta C_P \quad (a)$$

COMPATIBILITY:

$$\Delta L_R = \Delta L_P; \quad \frac{\Delta T_R L}{A_R E} = \frac{\Delta C_P L}{A_P E} \quad (b)$$

LEAKAGE REQUIREMENT:

$$\Delta C_P = 260 \text{ kN} \quad (c)$$

SUB (c) INTO (b):

$$\Delta T_R = 123.2 \text{ kN} \quad (d)$$

SUB (c) & (d) INTO (a)

$$\underline{P = 60.23 \text{ MPa}}$$

$$\underline{\Delta \sigma_R = \frac{123200}{706.9} = 174.3 \text{ MPa}}$$

$$\underline{\sigma_{R(FINAL)} = 542.1 \text{ MPa}}$$

1.26

(a) Figure a shows the bars subjected to force P .
 Figure b shows the free-body diagrams of the steel bar, the aluminum bar, and point A. By the free-body diagram of point A,

$$P = P_s + P_a \quad (a)$$

By Eq. (1.2) and Figs. a and b,

$$\delta_a = \frac{P_s L_s}{E_s A_s} = \frac{P_a L_a}{E_a A_a} \quad (b)$$

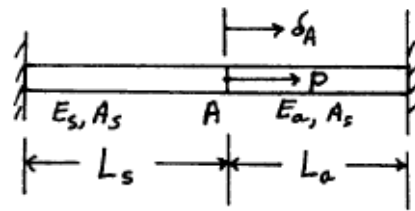
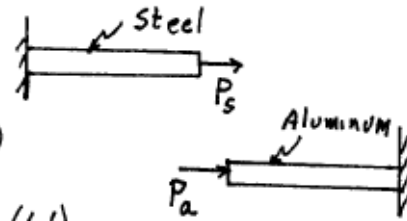


Figure a

By Eqs. (a) and (b),

$$P_s = \frac{P_a E_s A_s L_a}{E_a A_a L_s} = \frac{(P - P_s)(E_s A_s L_a)}{E_a A_a L_s}$$



Solving this equation for P_s , we find

$$P_s = \frac{P E_s A_s L_a}{E_s A_s L_a + E_a A_a L_s}. \text{ Hence, by Eq. (1.1),}$$

the stress in the steel bar is

$$\sigma_s = \frac{P_s}{A_s} = \frac{P}{A_s} \left(\frac{E_s A_s L_a}{E_s A_s L_a + E_a A_a L_s} \right)$$

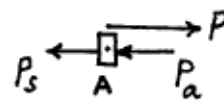


Figure b

Similarly, by Eqs. (a) and (b),

$$P_a = \frac{P E_a A_a L_s}{E_s A_s L_a + E_a A_a L_s} = \frac{(P - P_a)(E_a A_a L_s)}{E_s A_s L_a}$$

$$\text{or } P_a = \frac{P E_a A_a L_s}{E_s A_s L_a + E_a A_a L_s}$$

Hence, the stress in the aluminum bar is

$$\sigma_a = \frac{P}{A_a} \left(\frac{E_a A_a L_s}{E_s A_s L_a + E_a A_a L_s} \right)$$

(cont.)

1.26 cont.

(b) When the left wall is displaced to the right by an amount δ , the point A is displaced to the right by an amount δ_A (Fig. c).

By Eq. (1.2) and Fig. c,

$$\delta_A = \frac{FL_a}{E_a A_a} \quad (a)$$

and

$$\delta - \delta_A = \frac{FL_s}{E_s A_s} \quad (b)$$

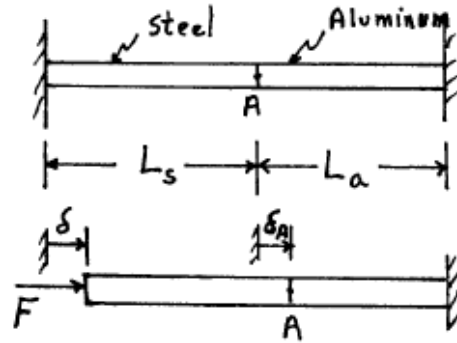


Figure C

By Eqs. (a) and (b),

$$\delta = \delta_A + \frac{FL_s}{E_s A_s} = F \left(\frac{L_a}{E_a A_a} + \frac{L_s}{E_s A_s} \right)$$

or

$$F = \frac{\delta E_a A_a E_s A_s}{E_s A_s L_a + E_a A_a L_s}$$

Hence, by Eq. (1.1), the stress in the steel bar is

$$\sigma_s = \frac{F}{A_s} = \frac{\delta E_a A_a E_s}{E_s A_s L_a + E_a A_a L_s}$$

and the stress in the aluminum bar is

$$\sigma_a = \frac{F}{A_a} = \frac{\delta E_a E_s A_s}{E_s A_s L_a + E_a A_a L_s}$$

1.31

The stress-strain curve is shown in Fig. a.

By Fig. a, the ultimate strength is $\sigma_u \approx 665$ MPa. Since each rectangular box under the stress-strain curve represents $(100)(0.05) = 5 \text{ MN}\cdot\text{m}/\text{m}^3$ of energy and we estimate that there are approximately 29.5 boxes under the stress-strain curve, we find

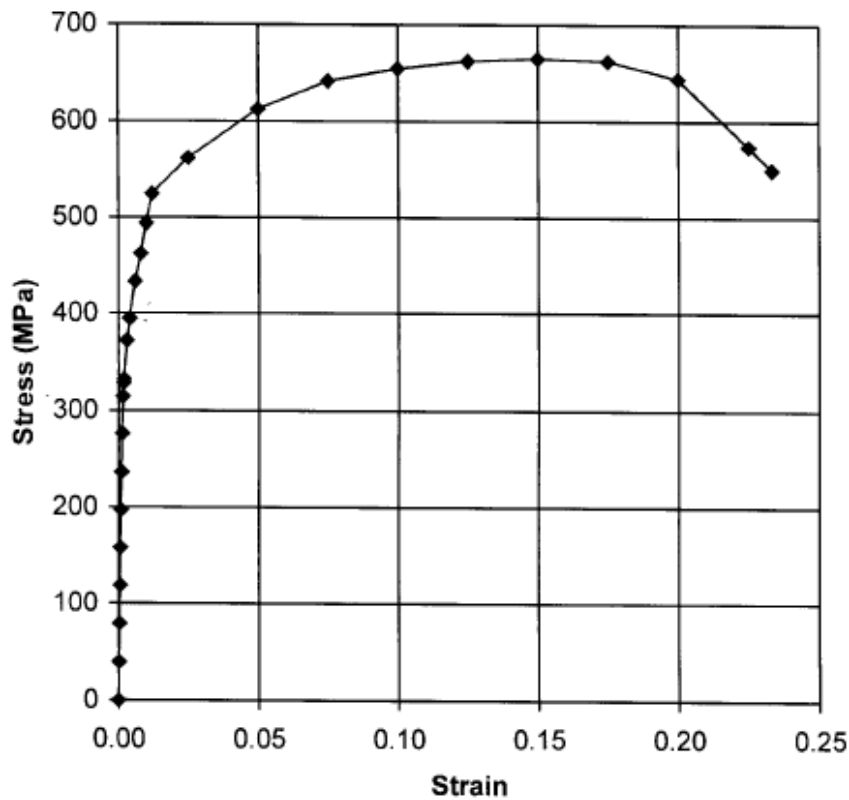


Figure a

that the modulus of toughness is

$$U_F = (5)(29.5) = 147.5 \text{ MN}\cdot\text{m}/\text{m}^3.$$

By numerical integration,

$$U_F = 144.2 \text{ MN}\cdot\text{m}/\text{m}^3$$

1.32

Extending the straight line portion of the stress-strain curve (Fig. a), we see that the slope of the line is

$$E = \frac{400}{.002} = 200 \text{ GPa.}$$

also, by Fig. a,

$$\sigma_{ys} = 395 \text{ MPa and } \sigma_{PL} = 315 \text{ MPa}$$

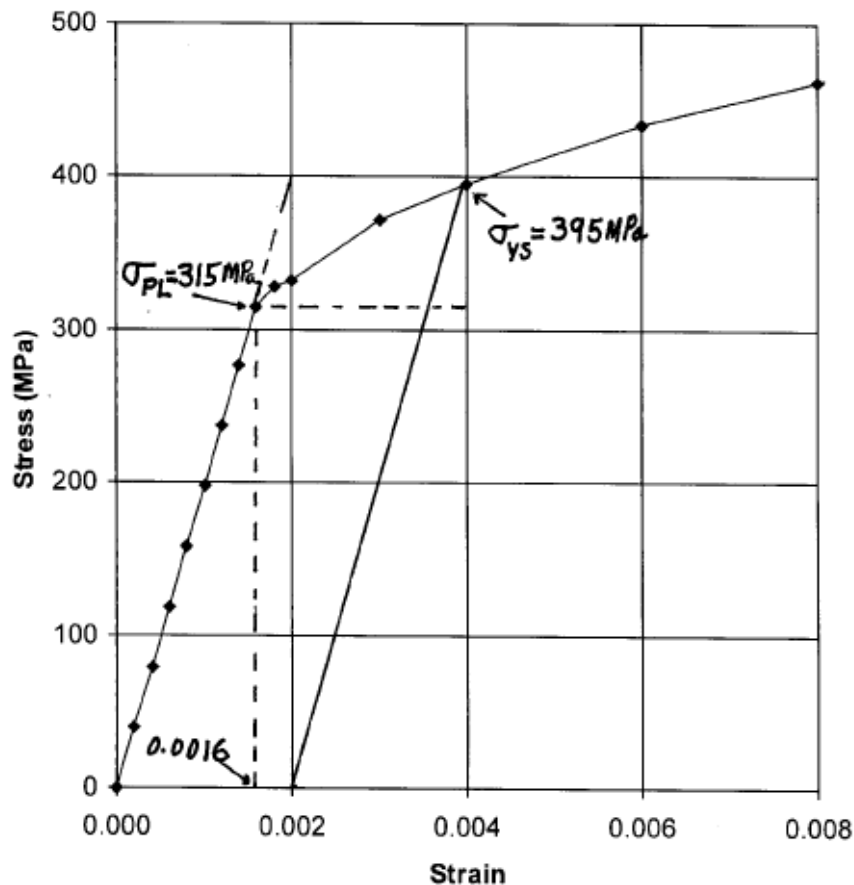


Figure a

The area under the stress-strain curve to the yield strength is approximately (see Fig. a)

$$\text{Area} = \frac{1}{2}(315)(0.0016) + (315)(0.004 - 0.0016) + \frac{1}{2}(395 - 315)(0.004 - 0.0016)$$

or,

$$\text{Modulus of Resilience} = 1.104 \text{ MN}\cdot\text{m}/\text{m}^3$$